

# ***Solving Problems and Saving Energy in Chilled Water Systems***

*Important considerations for owners and consulting engineers*

- Overflowing coils is responsible for low chilled water temperature differential (*Low Delta T*)
  - ✓ Inadequate control valves
  - ✓ Inaccurate instrumentation
  - ✓ Chilled water coil inlet and outlets reversed
- Chiller system inefficiencies and reduced capacity (*Due to Low Delta T*)
- Frozen cooling coils
- Lack of redundancy in supply and return fans
- Cooling tower basin sediment buildup
- Excessive fouling of chiller condenser water tubes
- Control of Legionella
- Corrosion of chilled water piping due to sediment, water quality, and microbial activity

**Times have changed.... Global Warming, etc.**

**Older systems were designed for low risk using proven simple designs. (Don't blame designers)**

**Today systems are more complicated and require more extensive and precision high performance engineering design.**

***High performance design does not mean higher first cost!!!***

*The majority of the time high performance reduces first cost by reducing the amount and the amount and/or size of equipment by precision engineering design.*

*High performance design reduces risk because system operation is greatly improved.*

## Typical Conditions

8 Row Coil

Face Velocity

Entering Water

Entering Air

Leaving Air

Air Pressure Drop

Water Pressure Drop

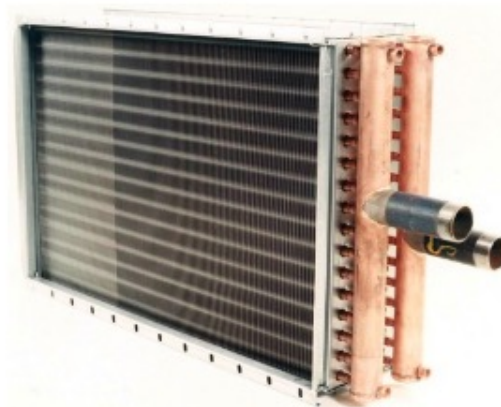
Fouling Factor

**30 Δ°F Chilled Water**

*Coil performance  
validated every four  
years by AHRI*

<b>Model Number</b>	DW0B55104G0FB140HABA0AB****
<b>System type</b>	Chilled Water W
<b>Rows</b>	8
<b>Tube matl/wall thickness</b>	.020 (0.508 mm) copper
<b>Nominal fin spacing</b>	140 fins per foot
<b>Fin material</b>	Aluminum
<b>Fin type</b>	Prima-flo H (Hi efficient) .01"
<b>Actual coil face area</b>	40.08
<b>Nominal coil height</b>	55" (1397 mm)
<b>Finned length</b>	104" (2642 mm)
<b>Casing option</b>	Galvanized
<b>Turbulators</b>	Yes
<b>Rigging weight</b>	1385.1 lb
<b>Installed weight</b>	1743.4 lb
<b>Tube matl/wall thickness</b>	.020 (0.508 mm) copper

Capacity	
<b>Total capacity</b>	761.20 MBh
<b>Sensible Capacity</b>	545.85 MBh
Air	
<b>Elevation</b>	700.00 ft
<b>Actual airflow</b>	20000 cfm
<b>Entering dry bulb</b>	80.00 F
<b>Entering wet bulb</b>	67.00 F
<b>Leaving dry bulb</b>	54.70 F
<b>Leaving wet bulb</b>	54.60 F
<b>APD</b>	1.032 in H2O
<b>Face velocity</b>	499 ft/min



Fluid	
<b>Standard fluid flow rate</b>	50.46 gpm
<b>Entering fluid temp</b>	40.00 F
<b>Leaving water temperature</b>	70.09 F
<b>Fluid PD</b>	6.38 ft H2O
<b>Fluid velocity</b>	1.51 ft/sec
<b>Fluid type</b>	Water
<b>Fouling factor</b>	0.00050 hr-sq ft-deg F/Btu
<b>Volume</b>	42.88 gal
<b>Reynolds number</b>	5942.92 Each
AHRI 410 Classification	
<b>AHRI 410 classification</b>	NOT certified by AHRI
<b>Data generation date</b>	8/31/2021
<b>Trane Select Assist update number</b>	2500

# Coil and Control Valve Selection for Proper Chiller System Loading

- Coils have been historically selected for much high chilled water flows than needed. Coils should be selected for lowest flow with acceptable (low) air and water pressure drops and  $\approx 30 \Delta^{\circ}\text{F}$ .
- Coils typically operate at two to three times the required flow because of control valve and other instrumentation limitations..
- Metering control valves are required for precision control in chilled water coils. Standard pressure dependent or valve/flowmeter control methods cannot control flow within the tolerance required wasting energy and increasing the potential for coil freezing.
- Chillers can be fully loaded and chilled system can be operated at optimal efficiency chilled water coil flow is precisely controlled

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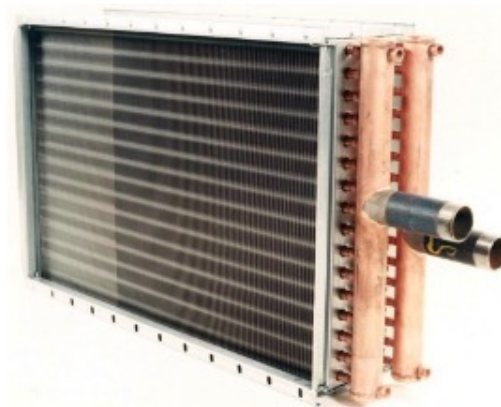
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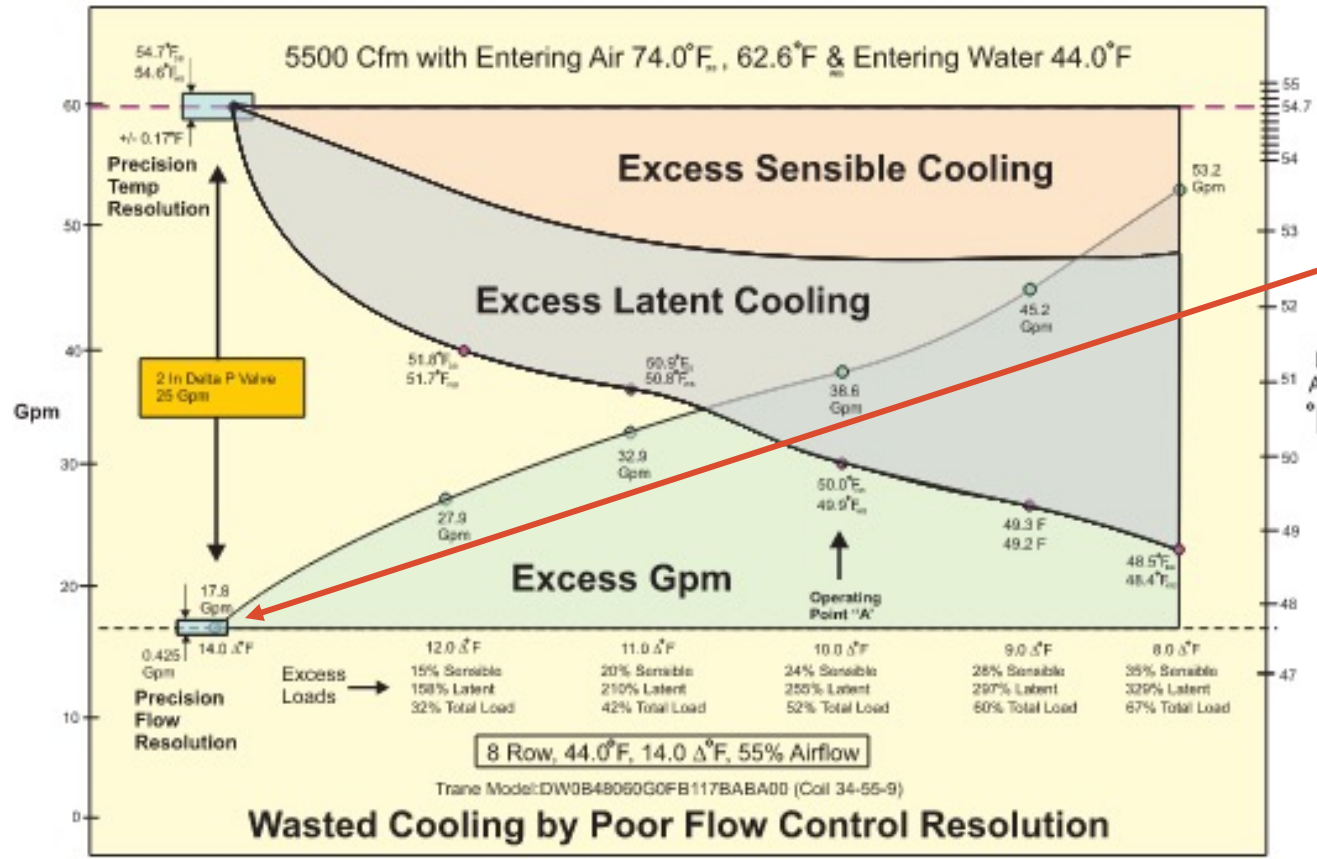
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*Precision metering valve with 0.425 gpm resolution will provide exacting control of coil leaving air temperature and maintain maximum possible Delta T*

W Coil, Water  
 8 Row; 117 Al Fins/Ft  
 10,000 Ft<sup>3</sup>/Min, 500 Ft/Min  
 Water: 44.0°F/58.0°F  
 Water: 53.96 Gpm  
 Water: 0.0005 Hr-Ft<sup>2</sup>-°F/Btu  
 Water: 2.46 Δ Ft-Wc  
 Air: 0.725 Δ In-Wc  
 Enter: 80.0°F<sub>db</sub>, 67.0°F<sub>wb</sub>  
 Leave: 54.7°F<sub>db</sub>, 54.6°F<sub>wb</sub>  
 1000 Ft Elevation  
 0.020 In Copper Tube

**Wasted Cooling by Poor Flow Control Resolution**

## Coil Leaving Air Dehumidification and Sensible Cooling Control

- During dehumidification the leaving air should be controlled at a dewpoint that maintains the space relative humidity setpoint.
- During sensible cooling operation (chiller on) the leaving air should be controlled at a drybulb temperature setpoint that minimizes energy consumption of the supply fan, reheat coil, and chiller system subject a lower limit imposed by duct insulation and other factors. In general, lowering the temperature setpoint to around 50 °F would be appropriate.
- Proper operation of leaving air temperature control requires that humidity and temperature instrumentation are calibrated and have the required accuracy and precision. Deficiencies in instrumentation are major factors in overflowing coils.
- High performance design determines the accuracy and precision of the temperature and humidity transducers and to know their calibration requirements.
- It is necessary to provide temperature and pressure test ports at coil inlet and outlet for calibration and testing.
- Sidestream chilled water filtration with monitor coupon racks are required for proper maintenance



Fan walls are applicable for new and retrofit air handling units eliminating single source failure.



Inlet



Discharge

## **Bypassed Chilled Water Piping & Reversed Coil Connections**

- It is not uncommon for chilled water coil supply and return piping to be reversed, which increases the chilled water flow and prevents proper cooling at higher cooling loads.
- Sometimes the chilled water supply and return pipes are cross connected due to field errors and is not caught during startup or commissioning.

## Inefficiencies & Load Shortfalls of Low Chilled Water Delta T

- Chilled water supply temperature is lowered to meet loads
- Additional chillers, chilled water pumps, condenser pumps, and cooling tower fans are operated to meet loads
- Primary/Secondary Chiller plants are unable to deliver rated cooling capacity
- Conversion of primary/secondary to variable flow primary with precision metering control valves during plant expansions will allow chilled water plants to fully load chillers

## Common Problems and Solutions

- Sediment buildup in cooling towers fouls heat transfer surfaces, reduces equipment life, fuels microbial growth, and requires excessive chemical treatment.
- Basin sweeper jets and “pod” type automatic backwashing filtration of 10-15% of cooling tower flow will control sediment in cooling towers.
- Water treatment with precision, calibrated instrumentation is critically important to limit fouling of chiller condenser tubes, control legionella, and protect cooling tower and piping.
- Condenser approach temperatures (refrigerant/entering condenser water) should be recorded and condenser tubes cleaned and eddy current testing as needed..

## Problem

- The 100 Ton air cooled chiller dedicated to a surgery rooftop unit was failing
- Existing water cooled 500 ton and 200 ton air cooled chillers were unable to deliver their rated capacity due to low chilled water Delta T (6 °F, Design 14 °F).
- Surgery cooling was dependent on a single chiller—No redundancy

## Solution

- The 100 Ton air cooled chiller was demoed
- Precision metering control valves were installed on all chilled water coils
- Precision RTD temperature transducers and temperature/pressure ports installed on all chilled water coils
- **Result:** 250 tons of additional capacity was reclaimed from 500 ton and 200 ton air cooled chillers with chilled water Delta T increasing close to design (13 °F). Chiller system had needed redundancy.



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